

## Article

# Research of Large Inflow Angles BEMT-Based Analytical–Numerical Performance Evaluation Model

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**Abstract:** This paper presents a comprehensive analytical–numerical algorithm constructed for propeller performance evaluation, focusing on accommodating large inflow angles. The algorithm’s design, range, and analytical features are clarified, indicating its potential to improve performance analysis, particularly for blades with substantial pitch variations. The Stahlhut model has not been validated against the conventional BEMT small-inflow angle methodology. This paper implements a modified Stahlhut model, coupled with the conventional BEMT. Preliminary validations of the model demonstrate promising results, with deviations reduced to  $-3\%$  to  $4\%$  compared to conventional BEMT methods exhibiting deviations as high as  $20\%$  to  $88\%$  against experimental data for a highly twisted propeller. The reconsideration of the computational module carries considerable implications for the design and refinement of propellers, providing alternative analysis methods that could improve operational effectiveness across a range of flight scenarios. Drawing upon the theoretical framework presented by Stahlhut, the algorithm enables a more complex understanding of propeller dynamics, facilitating accurate predictions of the loads at each blade section. The introduced algorithm emerges as a valuable asset for evaluating propeller performance during the early stages of design and certification, offering both low computational cost and medium to high reliability.

**Keywords:** BEMT; airfoil interpolation; propeller performance; inflow angle



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## 1. Introduction

The Blade Element Momentum Theory (BEMT) emerged as one of the pioneering modeling frameworks for propellers, offering a practical method to assess the performance of propellers and rotors, particularly in hover and axial flow conditions [1]. Widely utilized in the preliminary stages of propeller design [2,3], BEMT represents the standard approach for early design processes. To maintain reliability, the conventional BEMT methodology is built up with compressibility corrections [4], Prandtl’s tip-loss method [5], and considerations for rotational effects [6,7], aiming to address real-world complexities that may have been oversimplified in the original theory. More sophisticated techniques involve inviscid surface methods like panel methods coupled with free wake vortex lattice methods (FWVLM) or Computational Fluid Dynamics (CFD), which offer enhanced resolution of the overall flow field but incur higher computational costs [8,9].

Several adaptations have extended BEMT to account for non-zero incidence angles and nonuniform inflow around the azimuth [10]. Another approach, allowing BEMT to consider large inflow angles, has been proposed by Stahlhut [11], which serves as the theoretical foundation of this paper. Alternative methods like BET incorporating dynamic inflow models [1,12] offer precise aerodynamic forecasts for edgewise flight [13], delivering detailed flow-field data beyond the propulsor disk. This flexibility enables the assessment of diverse operational conditions and the exploration of geometric variables, facilitating the creation of internal optimization processes [14,15] to adjust system trimming or modify propeller geometry to meet specific performance criteria. The trimming is generally

executed by the RPM of the prop rotor and a combination of collective and feathering [16,17]. The possibility to assess the performance of prop rotors with low computational costs for varying geometrical definitions has become more relevant with the introduction of novel designs for emerging fields such as Urban Air Mobility (UAM) [18] and Unmanned Aerial Vehicles (UAV) [19] applications that operate at non-conventional flight regimes.

Conventional BEMT models, commonly found in the literature and low-to-medium fidelity tools, are constrained by simplifications in the numerical process, often requiring iterative methods for resolution [20]. These limitations stem from assumptions such as the insignificance of out-of-plane velocity ( $W_T$ ) compared to the in-plane velocity ( $U_T$ ), small induced angles, and the dominance of the local lift ( $dL$ ) over drag ( $dD$ ) components at each blade section [21]. In contrast, Stahlhut [11] introduced an alternative approach that discards small angle assumptions, explicitly includes in-plane velocity components, and does not assume negligible drag relative to lift, avoiding linearized simplifications and iterative procedures. Fundamentally, the conventional BEMT provides a simplified framework for comprehending the intricate dynamics of fluid–structure interaction. This framework involves a dual decomposition: radial separation of blades and the fluid column into blade elements. It includes a conceptual division between a macroscopic component following the Momentum Theory, and a local planar component adhering to the Blade Element Theory.

This paper aims to propose a prop rotor algorithm that combines, for a common set of inputs, a performance analysis following the conventional BEMT or an alternative solver using the Stahlhut [11] equation systems, which generalizes to large inflow angles. The algorithm is then able to assess the prop rotor performance based on predefined atmospheric conditions, prop rotor geometry, and airfoil selection. Then, the algorithm is validated by comparing the results from both the BEMT and Stahlhut solvers with third-party experimental data from prop rotor bench tests. This integrated approach seeks to provide a more accurate framework for evaluating prop rotor performance with an analytical–numerical method.

## 2. Analytical–Numerical Performance Model Architecture

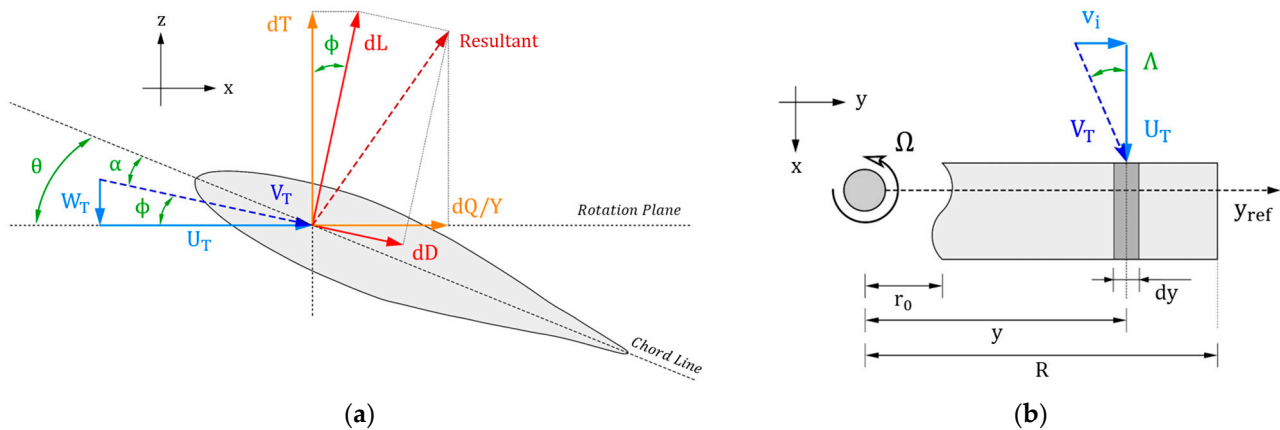
The diagram illustrating the sequence of modules in the algorithm under discussion is presented in Figure 1. Essentially, BEMT relies on extracting 2D static data, particularly the lift and drag of each blade element considered. Typically, airfoil polar coefficients are provided as tabulated data, directly sourced from tools such as XFOIL [22], tabulated data of airfoils, or more precise methods such as in-plane CFD computations [23]. Other methods, such as artificial neural networks fed by the CFD simulations, can be used to enhance the accuracy of the previously mentioned stall delay models [14]. The lift and drag of each element can be derived using these known aerodynamic coefficients, considering the dynamic pressure and the element’s chord. Generally, all the prop rotor analytical–numerical 2D methods contain the following limitations [24]:

- The flow is incompressible, inviscid, irrotational, and uniform;
- There is a continuous flow velocity and pressure, except at the disk;
- The airfoils through the blade do not interact between them;
- The blades of a prop rotor do not interact between them either.

A set of non-dimensionalized parameters is defined and used to evaluate the prop rotor system’s performance results during the intermediate calculation process. The graphical determination of the convention is given in Figure 1. The pitch ( $\theta$ ), which can be regulated by the collective and feathering mechanism, is considered the sum of the angle of attack ( $\alpha$ ) and the inflow angle ( $\phi$ ).

The primary distinction between the BEMT and the Stahlhut solvers lies in calculating the inflow angle, which subsequently determines the angle of attack. The system of equations of the Stahlhut solver, which does not neglect large inflow angles, is presented in Equations (1)–(5). The description of the symbols presented in the equations is given in Table 1. The flowchart of the proposed algorithm for conventional BEMT and the Stahlhut

solvers is given in Figure 2. Once the angles of attack and the total velocities, i.e., the Reynolds, have been calculated, they are integrated into the prop rotor performance parameters according to the conventional BEMT methodology. The lift and drag coefficients are extracted from the airfoil performance database, and the general performance parameters can be calculated for each blade section and integrated throughout the blade span.



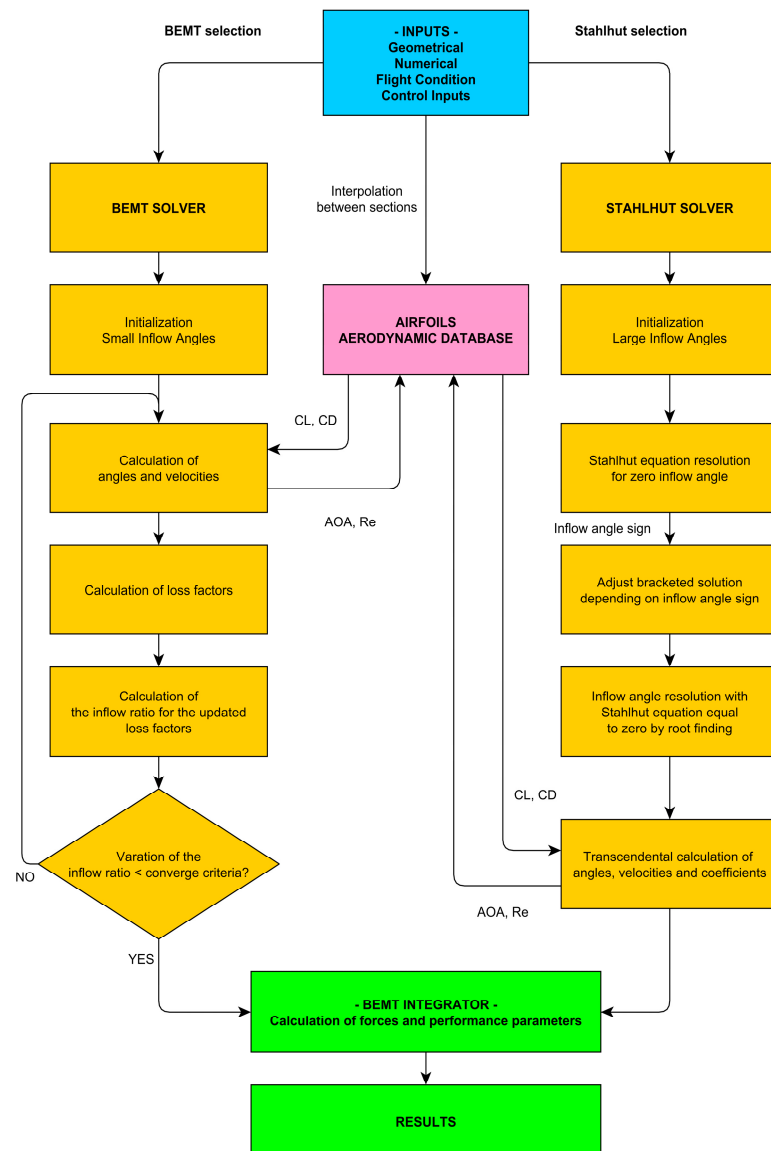
**Figure 1.** Blade element sign convention from the lateral (a) and top view (b).

**Table 1.** Description of the presented mathematical symbols.

Symbol	Description
$\Omega$	Prop rotor angular velocity
$y$	Absolute blade element position
$\phi$	Inflow angle
$V_\infty$	Free-stream velocity
$\sigma$	Solidity ratio
$C_L$	Lift coefficient
$C_D$	Drag coefficient
$\gamma$	Drag-to-lift coefficient ratio
$r$	Relative element position
$K_T$	In-plane loss factor
$K_P$	Out-of-plane loss factor
$F$	Prandtl loss factor
$C_T$	Thrust coefficient
$C_P$	Power coefficient
$\xi$	Swirl velocity ratio
$\lambda$	Total inflow ratio
$\lambda_i$	Induced inflow ratio

The proposed framework provides the algorithm user with the option to choose between two solvers: the conventional BEMT solver, which disregards large inflow angles, and the Stahlhut solver, which accounts for them. Both methods share a common starting point, as the algorithm is initialized with the same data as referred to in the first cell of Figure 2. The results of the algorithm are presented in a consistent format, where due to differences in the computational approach, the numerical values of each solver are expected to vary. This paper evaluates these differences in the prop rotor performance algorithm for a specific prop rotor configuration.

The relative position element ( $r$ ) is a parameter that defines the position of each blade section. This parameter is defined as the ratio between “ $y$ ” and “ $R$ ”, which is used for integrating the performance parameters throughout the blade span. The parameter “ $y$ ” defines the absolute position of the section, which is then normalized with the blade radius “ $R$ ”. The root cutout is denominated as “ $r_0$ ”.



**Figure 2.** Flowchart of the prop rotor performance model.

The Stahlhut system cannot be solved analytically; hence, it must be solved numerically. The convergence of the system is not guaranteed due to the presence of nonlinear, transcendental equations. The recommended method is to express the system in terms of  $\phi$  and solve it using a bracketed method such as the bisection method, realizing that a single solution is located within a range between two points.

$$g(\phi) = (\Omega y \sin(\phi) - V_{\infty} \cos(\phi)) \sin(\phi) - \operatorname{sgn}(\phi) \frac{\sigma C_L \sec(\gamma)}{8r} \left[ \frac{\Omega y}{K_T} \cos(\phi + \gamma) + \frac{V_{\infty}}{K_P} \sin(\phi + \gamma) \right] \quad (1)$$

Here,  $K_T$  represents the in-plane loss factor, while  $K_P$  denotes the out-of-plane loss factor. These factors are determined using the Formulas (2) and (3), respectively. The Prandtl loss factor, “ $F$ ”, is derived from the earlier-defined equation concerning the loss factor for the allowance to large inflow angles approach. Consequently, unlike the BEMT approach, there is no need for an iterative process.

$$K_T = 1 - (1 - F) \cos(\phi) \quad (2)$$

$$K_P = 1 - (1 - F) \sin(\phi) \quad (3)$$

The methodology for the large-angle approach remains the same as that of BEMT, but the system is more complex mathematically. The thrust and the power coefficient equations take the form of the Formulas (4) and (5), respectively.

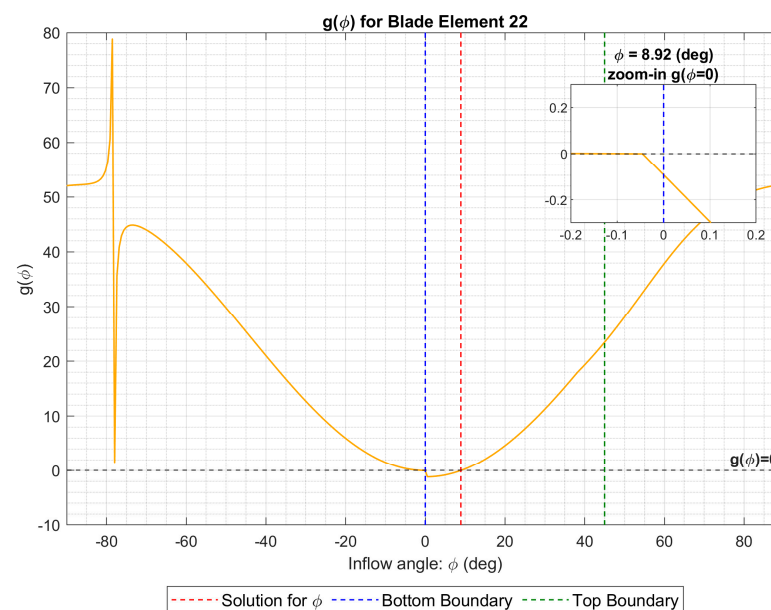
$$dC_T = \frac{1}{2} \sigma \sqrt{\xi^2 + \lambda^2} (C_L \xi - C_D \lambda) dr = 4 K_T |\lambda| \lambda_i r dr \quad (4)$$

$$dC_P = \frac{1}{2} \sigma \sqrt{\xi^2 + \lambda^2} (C_L \lambda + C_D \xi) r dr = 4 K_P |\lambda| \xi r^2 dr \quad (5)$$

The total velocity can be calculated in two different ways, depending on whether the conventional BEMT method or the Stahlhut-based methodology has been chosen. It can be observed how the differences rely on the in-plane velocity implementations, whereas the velocity normal to the rotor disk remains the same.

The  $V_\infty$  is defined as the velocity normal to the disk. The presented transcendental equation [11] considers no incidence angles of the incoming flow into the disk. The azimuth angle of each blade section with respect to the incoming flow would need to be evaluated, leading to time-variant performance results.

The sharp oscillation of  $g(\phi)$  with respect to  $\phi$  during its calculation, as given in Figure 3, is due to the reach of a singularity section. In cases with an extensive range of inflow angles, multiple solutions to  $\phi$ , one positive and another negative, might be found. To determine the correction solution,  $g(\phi)$  is first calculated for  $\phi = 0$ . If  $g(\phi) > 0$ , then the  $\phi$  is negative, whereas if  $g(\phi) \leq 0$ , the  $\phi$  is positive. The boundaries for the bracketed solution are adjusted accordingly by a conditional statement.



**Figure 3.** Calculation of the inflow angle by the bisection method for a particular blade element.

To advance the applicability of the prop rotor performance model, it is recommended to include non-axial flow components derived from the angle between the disk plane and  $V_\infty$ . This becomes particularly relevant during transition phases in VTOLs, where the ability to account for non-axial flow can provide a more accurate representation of prop rotor behavior. However, it is to be noted that up to incidence angles of 12–15°, no noticeable improvements in the prop rotor performance modeling are expected to be found.

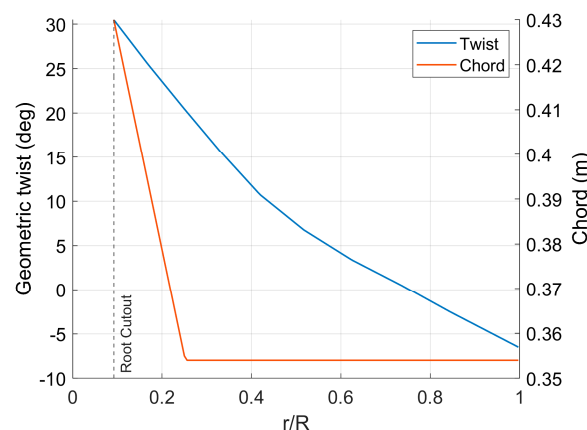
### 3. Specification of System Configuration

The validation of this study is based on the officially reported Bell XV-15 prop rotor parameters. The XV-15 is an advanced high-wing tiltrotor aircraft with dual tandem

proprotors near the wingtip. This aircraft is commonly known as the predecessor of the Bell V-22 Osprey. The experimental results are taken from the bench tests performed by NASA at the Ames Research Center, with the reference Technical Memorandum 86-833 [25]. The necessary parameters for the validation are the geometrical definition of the proprotor, airfoil selection, the RPM settings, experimental atmospheric conditions, and the measured thrust and power coefficients. Further information about these inputs is provided in the paragraphs below.

The proprotor system is defined primarily by the number of blades, the radius, root cutout, the chord and twist vectors [26], and the airfoil determinations throughout the blade span. The technical specifications of the Original Equipment Manufacturer (OEM) aircraft [27] outline the characteristics of the proprotor, including geometric twist, root cutout, and chord distribution along the blade span. The geometrical twist, or chord line angle, is the pitch measured from the chord line to the rotational plane. The chord has been presented in absolute values, even though some authors might represent it as the ratio between the chord value and the radius of the rotor blade. The root cutout is the position at which the blade is considered to start, measured from the center of rotation.

The collective is neglected in the definition of the geometrical twist, which would typically be considered the rest position. The root cutout aims to mitigate potential reverse flow effects near the root since the region is over the hull. These parameters are presented for the study case in Figure 4. With a geometric twist from  $30.5^\circ$  to  $-6.5^\circ$  from the root cutout to the blade tip ( $r/R = 1.0$ ), this study case is an excellent example of the potential implementation of large inflow angles in a proprotor performance tool.

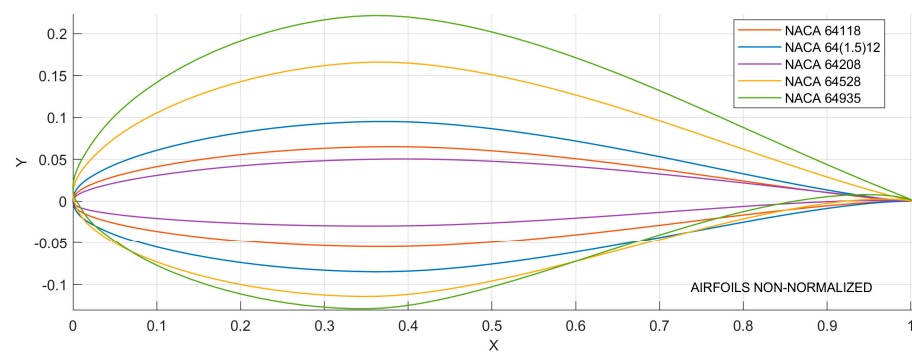


**Figure 4.** Geometric twist and chord of the blade.

The blade span, as well as other non-rotary wings, are generally composed of multiple airfoils. In the case of rotary wings, higher thickness-to-chord ratios are found closer to the root since the total velocity at the section is considerably lower than at the tip. Additionally, higher bending moments appear closer to the root, hence the need to increase the inertia modulus to compensate for these mechanical loads. The representation of the airfoil profiles for the reference study case and their relative position is given in Figure 5. The airfoils presented are taken in accordance with the XV-15 data [25].

Linear interpolation between the airfoil aerodynamic coefficients at sections between the exact positions for each could be performed. In this way, blade section polars are taken directly from the original database of each airfoil for those blade elements that contain the " $r/R$ " position inside its domain. This methodology gives each blade section a unique polar data frame, considering there are no identical consecutive airfoils. The relative location of each airfoil, given by the OEM [25], is given in Table 2.





**Figure 5.** Blade's airfoil representation.

**Table 2.** Airfoil references and relative positions throughout the blade span.

Airfoil	$r/R$
NACA 64118	1.00
NACA 64(1.5)12	0.80
NACA 64208	0.51
NACA 64528	0.17
NACA 64935	0.09

The airfoil aerodynamic coefficients, considering attached flow conditions and various other limitations, are recommended to be generated by the free-use XFOIL [22] program. The profiles with higher thickness-to-chord ratios present difficulties using this program to achieve the aerodynamic coefficients. Additionally, these airfoils are placed at the blade's lower speed region, accentuating the deattached flow phenomena. A smoothing method is recommended to compensate for some of the inaccuracies of the polar generation.

The airfoil aerodynamic coefficients have been extrapolated beyond the XFOIL limits to account for high angles of attack, which might be found close to the root and a high-speed flight. Several stall delay models take the dynamic stall effects into account. The most used ones are the Viterna-Corrigan post-stall model [28] and the Corrigan-Schilling [29] stall delay model. The first of the extrapolation models is available in the free-use QBLADE program, which was used to perform this particular task.

The presented propotor algorithm is a versatile tool capable of being adjusted so that the inputs become the performance parameters, resulting in optimized geometrical and airfoil definitions. The potential iterative refinement process would involve a continuous dialog between performance objectives and design parameters. For instance, the algorithm might prioritize maximizing thrust during takeoff, leading to adjustments in geometrical configurations and airfoil selections to achieve the desired outcome. Similarly, when focusing on reducing power requirements for the cruise, the algorithm may adjust configurations to minimize the power required for a given thrust and other flight condition parameters, such as the pressure altitude. Each iteration balances the predefined performance goals, operational constraints, and environmental considerations, resulting in a fine-tuned propotor design to meet the desired operational demands.

## 4. Validation of the BEMT and Stahlhut Models

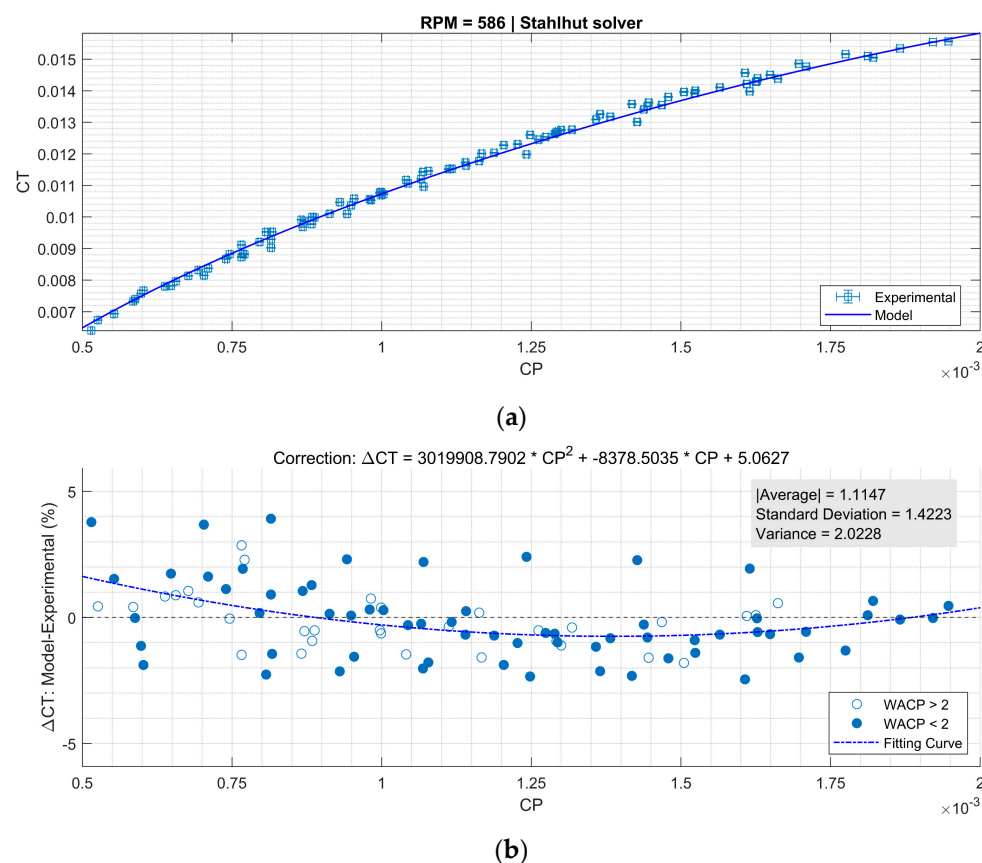
### 4.1. Static Thrust Evaluation

The static thrust is defined as zero forward speed related to the hovering or takeoff phases. Hover performance is critical to analyze since it usually determines the aircraft's maximum payload, especially in vertical flight operations. Additionally, the maximum thrust available in the case of propotors is generally encountered for the flight condition of zero velocity. Further details about this last statement are described in later sections of this project.

The evaluated bench-tests [25] were conducted outdoors, without aerodynamic interference from other elements and out of ground effect. The test rig and its supporting structure provide negligible blockage of the rotor wake. It has been reported that the prop rotor is sufficiently separated from the ground to be considered an out-of-ground effect (OGE).

The thrust balance is accurate within  $\pm 0.1\%$  error up to 50 kN, with no significant interactions caused by other forces or moments. The instrumented drive shaft torque is accurate to within  $\pm 0.3\%$  error up to the maximum capacity of 28.5 kNm. The tests have been performed with winds equal to or less than 1.5 m/s. The measured rotor torque has been corrected for the effect of wind using an empirical momentum theory-based correction procedure, for which further detail is available in the referred Technical Note [25]. The magnitude of the correction during the performed tests, at less than 1.5 m/s, is below 3%. This correction percentage is denominated in this document as WACP: Wind Adjusted Power Coefficient.

The experimental bench tests have been grouped into RPM sets with 511, 553, 565, 586, and 624 values. The data have been grouped so that the maximum deviation is four revolutions, which is considered to have a negligible effect on the results. The number of data points for each RPM set is 7, 4, 4, 168, and 8, respectively. Therefore, the population is considered enough for statistical analysis, just for the RPM set 586. The results of the Stahlhut solver for the RPM 586 are given in Figure 6.



**Figure 6.** Model and experimental  $C_T$  versus  $C_P$  absolute (a) and deviation (b) results for the Stahlhut solver and RPM = 586.

Considering an average air density value of  $1.235 \text{ kg/m}^3$ , an equivalent pressure altitude (ZP) of  $-80 \text{ m}$  has been configured for all the model computations. The progression of  $C_T$  and  $C_P$  is achieved by increases in the collective, which is aligned between the experimental and the model configuration in the case of the Stahlhut solver. For the BEMT solver, higher collective angles have been needed to match the  $C_P$  values.



A second-order polynomial equation, graphically represented by the name “Fitting Curve”, adjusts the error of  $\Delta C_T$  throughout the  $C_P$  range. Fitting the deviation of  $\Delta C_T$  With such a methodology, the physical meaning of the errors between the model and the experiments might not be captured. However, once the experimental data are known, this fitting curve is considered useful for a smooth adjustment of the model results. General parameters, such as the absolute average, standard deviation, and variance, are calculated for the statistical evaluation purposes of these deviations.

Table 3 presents the main statistical parameters: the absolute average, standard deviation, and variance for each RPM setting and selected solver. The absolute average is presented to account for the positive and negative deviations of  $\Delta C_T$ .

**Table 3.** Main statistical parameters from the deviation in thrust coefficient versus power coefficient.

Solver	RPM	Samples	Average	Standard Deviation	Variance
BEMT	511	7	56.07	15.95	254.44
	553	4	54.40	14.74	314.73
	565	4	53.44	17.59	309.43
	586	168	54.28	16.33	266.58
	624	8	50.74	16.67	277.97
Stahlhut	511	7	1.64	1.71	2.92
	553	4	1.35	1.74	3.03
	565	4	1.12	1.17	1.38
	586	168	1.11	1.42	2.02
	624	8	0.77	0.68	0.46

The analysis of the static thrust evaluation reveals intriguing relationships between solver choice, RPM settings, and key statistical parameters such as average, variance, and  $\Delta C_T$  deviations. Across both BEMT and Stahlhut solvers, noticeable trends emerge. For instance, examining the average  $\Delta C_T$  values, it is noticeable that the Stahlhut solver consistently yields significantly lower deviations than BEMT across all RPM settings. This suggests a potentially superior accuracy of the Stahlhut model in predicting thrust performance. Furthermore, as RPM increases, there appears to be a slight decrease in average  $\Delta C_T$  for both solvers, albeit with some variability. This trend could be attributed to improved model convergence at higher RPMs and Reynolds numbers.

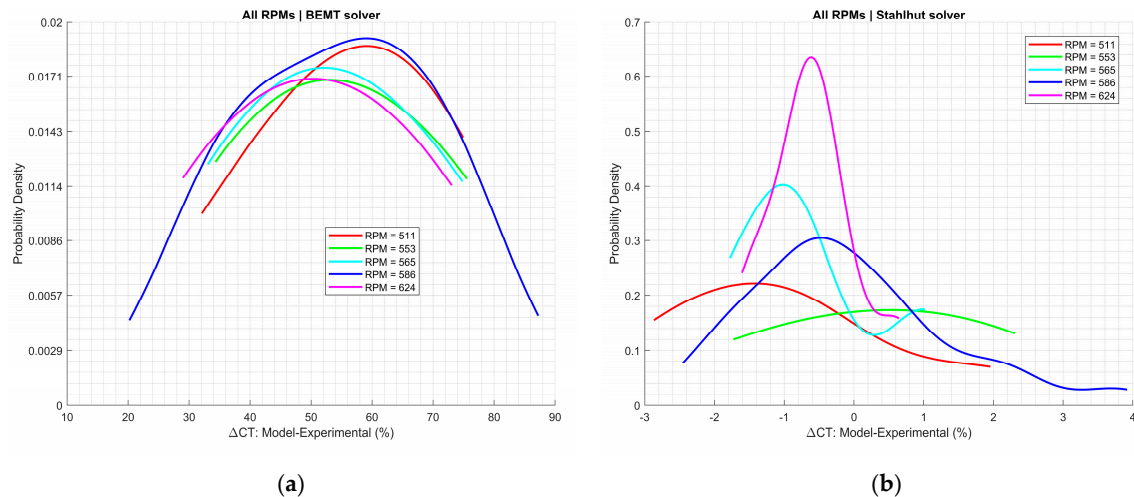
The relationship between RPM and variance cannot be clearly defined. While the variance tends to fluctuate across different RPM settings, it is interesting to note that for the BEMT solver, there is a general increase in variance as RPM rises. This could indicate increased dispersion or variability in model predictions at higher rotational speeds. In contrast, the variance for the Stahlhut solver shows less consistent patterns across RPM settings, suggesting potentially different sources of variability or model behavior.

Additionally, examining  $\Delta C_T$  alongside variance provides insights into the consistency and reliability of model predictions. Generally, higher variance values may imply more significant uncertainty or inconsistency in the model’s performance, potentially leading to larger deviations from experimental data. Interestingly, while the Stahlhut solver exhibits lower deviations overall, its variance values are not consistently lower than BEMT, indicating a nuanced relationship between accuracy and variability.

Considering the varying sample sizes across RPM settings adds a layer of complexity to the analysis of statistical parameters. The smaller sample sizes for specific RPM settings, such as 511, 553, and 565, could potentially introduce more significant variability and uncertainty in the calculated averages, variances, and deviations. For instance, the relatively small sample size for the 553 and 565 RPM settings might contribute to the observed higher variances compared to other settings. With fewer data points available for analysis, there may be less precision in estimating the true variability of the data. Conversely, the larger sample size for the 586 RPM setting allows for more robust statistical analysis, potentially resulting in more stable average, variance, and deviation estimates.

#### 4.2. Sensitivity Analysis

The probability density distribution (PDD) becomes a valuable method to analyze the likelihood of a particular  $\Delta C_T$  to be found at each RPM setting. The area enclosed by each probability density curve, the maximum values, and the relative position with respect to  $\Delta C_T$  are presented in Figure 7 for the BEMT and the Stahlhut solver, respectively.



**Figure 7.** PDD compilation between the model and experimental results for the BEMT (a) and Stahlhut (b) solvers.

The similarity in the shape of the PDD curves for the BEMT solver suggests that the performance of this solver is relatively consistent across different RPM settings. The wider range of  $\Delta C_T$  values for the BEMT solver, from 20 to 88% (RMS 68%), compared to the Stahlhut solver, from  $-3$  to 4% (RMS 7%), clearly represent the error ranges between the small and large inflow angle modeling approaches. The deviations in order of magnitude are similar to previous studies, also based on the Stahlhut methodology, for the same proprotor, with RMS errors of 6.16% between the  $C_T$  and  $C_P$  for Mach 0.6 (RPM = 586) [30]. The other RPM settings have not been validated by the previously mentioned reference. Additionally, the algorithm configuration was not disclosed, which made the comparison between the approach of this paper and the mentioned reference difficult.

These findings suggest that the BEMT solver tends to be more precise than the Stahlhut solver, but this second one is notoriously more accurate. Looking toward the implementations of these numerical-analytical methodologies, BEMT could be recommended to analyze trends in the design changes in proprotors. In contrast, the Stahlhut solver could be more suitable for quantifying the performance values of each design in the subsequent design validation loops.

Overall, the Stahlhut solver is more conservative than the conventional BEMT, especially closer to the tip and at higher collective angles. In other words, higher  $C_P$  and lower  $C_T$  tend to be found when considering the allowance for the large inflow angles approach. Except for the induced velocity, the rest of the parameters tend to maintain a certain parallelism between the BEMT and the Stahlhut results. The Reynolds throughout the blade span for both solvers and collective angles remain practically unchanged, as the primary contributor to this parameter is the angular velocity, which is fixed for this analysis.

Conventional BEMT solvers tend to have a linear progression of deltas between experimental and model results. These models are not recommended in the case of tiltrotor configurations, which possess great twist and speed gradients from root to blade tip. These conventional methodologies might be considered attractive if low computational costs are desired and an analysis of trends—differences between different proprotor geometrical configurations and operational conditions—is desired. The absolute values of the proprotor

performance would lead to misleading conclusions in the case of propotor configurations such as the ones considered in this study.

Expanding the validation loops to encompass a broader range of propotors' geometries is recommended. This would serve to verify the reliability and robustness of the propotor performance tool across various geometrical configurations and operating regimes. Incorporating additional propotors into the validation process will further enhance the tool's applicability and build confidence in its accuracy in architectures with not so accentuated pitch differences or greater RPMs.

## 5. Conclusions

1. The integration of allowances for large inflow angles significantly improves computational accuracy, substantially reducing deviations between model predictions and experimental results. With deviations ranging from  $-3\%$  to  $4\%$  (RMS  $7\%$ ), the large inflow angle model outperforms the small inflow angle approach, which exhibits deviations as high as  $20\%$  to  $88\%$  (RMS  $68\%$ ). The reduction in deviations observed with the large inflow angle model underscores its efficacy in capturing the intricacies of real-world operating conditions, instilling greater confidence in the predictive capabilities of the algorithm.
2. The analytical-numerical Stahlhut solver emerges as a promising tool in the development process of propotors, particularly those characterized by significant pitch variations along the blade span. Lower computational costs than alternative methods such as FVM or CFD can be achieved without significantly compromising accuracy. This capability streamlines the design process and opens opportunities for exploring a broader range of design possibilities, facilitating optimization cycles in the propotor development process.
3. While the BEMT method remains valuable for analyzing design trends and preliminary assessments of propotor configurations, the Stahlhut solver offers a more accurate approach for quantifying performance values in subsequent design validation loops. This strategic division of roles capitalizes on the strengths of each methodology, ensuring comprehensive evaluations of propotor designs while optimizing computational resources and enhancing overall design efficacy for numerical-analytical processes.

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